**Original** Article

# Towards a Novel Hybrid Fuzzy Logic-Based Control Strategy to An Inverted Pendulum System

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**Abstract** - The current study proposes a new solution with a successful control method to balance an inverted pendulum placed on a cart. The control plant is a nonlinear drive system consisting of a freely rotating rod and a small moving 4-wheel vehicle. A novel control strategy is created as a reasonable integration between a fuzzy logic structure built up depending on the Lyapunov stability theory and the Particle Swarm Optimization (PSO) algorithm. The Lyapunov theory has long been known for designing effectual control schemes. Meanwhile, the PSO mechanism, one of the most famous and efficient optimization methods, determines three scaling factors in the control diagram. Various simulation results of the proposed hybrid fuzzy logic controller in four scenarios have been successfully obtained using MATLAB/Simulink software, outperforming several existing counterparts, namely conventional PID, PD- and PI-like fuzzy logic regulators. Promising findings in this study verify the applicability of the proposed control methodology in both theoretical and practical aspects.

Keywords - IPBCS, Fuzzy logic, Lyapunov theory, PSO, Hybrid controller.

# **1. Introduction**

The control of an inverted pendulum balancing system for various desired positions of the pendulum bar involves harmonizing two regulating aspects: the pendulum rod and the four-wheeled moving vehicle [1-5]. The control method proposed in this study proves valuable for managing diverse nonlinear systems, such as robots and rocket propulsion [6, 7]. Traditionally, the Inverted Pendulum Balancing Control System (IPBCS) has been frequently employed to validate numerous significant control methods [8-11].

In recent years, several increasingly efficient controllers have been built upon fuzzy logic techniques, relying on conventional regulators like Proportional-Integral (PI), Proportional–Derivative (PD), and Proportional-Integral-Derivative (PID). For variable-timing systems with multiple factors featuring nonlinearities and uncertainties, fuzzy logic controllers with an inference system based on operational engineers' or experts' knowledge and expertise are more suitable than traditional counterparts such as PI, PD, and PID. The set of fuzzy rules draws on the experience and understanding of operational experts and testers accumulated over time to manage system operations in alignment with desired goals. Understanding the system enables us to operate and control fuzzy logic-based systems most effectively [12-17].

This research applies fuzzy logic based on the Lyapunov stability theory and the Particle Swarm Optimization (PSO) algorithm to design an intelligent controller for balancing the inverted pendulum system. As the inverted pendulum system is a nonlinear Multiple-Input and Multiple-Output (MIMO) system, the balancing control problem must be tailored to the system, utilizing the Lyapunov stabilization method to determine the fuzzy logic controller parameters. Technically, the fuzzy logic set comprises two inputs: the output feedback response and the output feedback derivative. This set produces three scaling factors which directly influence the transient and stability capabilities of the system. Three meaningful coefficients are determined using an appropriate optimal method. Metaheuristic optimization techniques such as Genetic Algorithm (GA) and PSO mechanism can successfully determine these three parameters [10]. The current study selects the PSO mechanism for this task due to its simplicity and effectiveness.

The PSO algorithm operates by setting an initial value for the system and then assigning elements to search for the best parameters within a given interval. The search is repeated in several sets, updating the values until the set condition is satisfied. Finally, the best parameters that meet the terminating requirements can be obtained. These optimal factors can be used to design an efficient control methodology. The subsequent sections of the current paper are organized as follows. After the Introduction section, section 2 introduces the model and formula of the driven inverted pendulum system. This is followed by Section 3, which discusses the controllers used and outlines how to construct a dimming controller based on Lyapunov's stability theory. A novel hybrid control strategy is also presented in this section. Section 4 introduces numerous scenarios of numerical simulation to validate the proposed controller. Finally, Section 5 provides conclusions and outlines future work inspired by this study.

## 2. Modelling the Inverted Pendulum System

The inverted pendulum model is mounted on a cart that can move along a rail, as shown in Figure 1. It is supposed to ignore friction between the cart, the road, and the cart and the pendulum. The parameters of the system, as indicated in Table 5 of the Appendix, include M - the mass of the vehicle; m - the group of the rod; l - length of the rod; x(t) - location of the vehicle;  $\theta(t)$  - rotational angle joined by the rod to the vertical plane; u(t) - the driving force.



Fig. 1 A typical cart-mounted inverted pendulum model



Fig. 2 A diagram of the forces acting on the inverted pendulum system

It is necessary to describe the dynamic characteristics of the inverted pendulum system based on Newton's laws of motion. Mechanical systems have two axes: the movement of the pendulum car above the x-axis and the rotation of the pendulum rod on the xy plane. Analyzing the dynamics of the inverted pendulum system, a diagram of the forces acting on the cart and the pendulum bar is plotted in Figure 2.

It is possible to sum up the forces vertically. Still, they are not helpful because the motion of the inverted pendulum system does not move in this direction, and the Earth's gravity is in balance with all the vertical forces. Hence, it is reasonable to establish a system of equations describing the nonlinear kinematic properties of the inverted pendulum system based on the *x*-axis as follows:

$$(M+m)\ddot{x} + b\dot{x} + ml\ddot{\theta} - ml\dot{\theta}^2\sin\theta = u$$
(1)

$$(J+ml^2)\ddot{\theta}+m\lg\sin\theta=-ml\ddot{x}\cos\theta$$
 (2)

The above two equations can be transformed as:

$$\ddot{x} = \frac{u - b\dot{x} - ml\ddot{\theta}\cos\theta + ml\dot{\theta}^2\sin\theta}{M + m}$$
(3)

$$\ddot{\theta} = \frac{-ml\ddot{x}\cos\theta - m\lg\sin\theta}{J + ml^2}$$
(4)

From (1)-(4), the mathematical equations of the nonlinear inverted pendulum are obtained below:

$$\ddot{x} = \frac{\left(J + ml^{2}\right)\left(u - b\dot{x} - ml\dot{\theta}^{2}\sin\theta\cos\theta\right)}{\left(J + ml^{2}\right)\left(M + m\right) - m^{2}l^{2}(\cos\theta)^{2}} + \frac{m^{2}l^{2}g\cos\theta\sin\theta}{\left(J + ml^{2}\right)\left(M + m\right) - m^{2}l^{2}(\cos\theta)^{2}}$$
(5)

$$\ddot{\theta} = \frac{ml\left(b\dot{x}\cos\theta - u\cos\theta - ml(\cos\theta\sin\theta)\dot{\theta}^{2}\right)}{\left(J + ml^{2}\right)\left(M + m\right) - m^{2}l^{2}(\cos\theta)^{2}} + \frac{ml(M + m)g\sin\theta}{\left(J + ml^{2}\right)\left(M + m\right) - m^{2}l^{2}(\cos\theta)^{2}}$$
(6)

To simplify the system of ignoring the mass to be shaken, the linear mathematical model of the inverted pendulum subsystem is defined as follows:

$$\ddot{x} = \frac{u + ml\dot{\theta}^2 \sin\theta - mg\cos\theta\sin\theta}{M + m - m(\cos\theta)^2}$$
(7)

$$\ddot{\theta} = \frac{u\cos\theta - (M+m)g(\sin\theta) + ml(\cos\theta\sin\theta)\dot{\theta}^2}{ml(\cos\theta)^2 - (M+m)l}$$
(8)

Two equations expressed in (7) and (8) can be used to design control schemes of an IPBCS.

# 3. The Controllers Used for the IPBCS

# 3.1. Traditional PID Controller

It should not be denied that PID controllers have been widely used in control systems, especially in factories. The PID controller calculates the error value as the difference between the variable parameter and the desired setpoint. Such a PID controller aims to minimize this error by adjusting the input value. The PID controller's calculation relies on three separate parameters: Proportional P, Integral I and Derivative D. The typical formula representing the operation principle of the PID with three scaling factors P, I and D is as follows:

$$u(t) = \underbrace{P.e(t)}_{\text{Proportional}} + \underbrace{I.\int e(t)dt}_{\text{Integral}} + \underbrace{D.\frac{de(t)}{dt}}_{\text{Derivative}}$$
(9)

#### 3.2. Traditional Fuzzy Logic Controllers

Traditional PID controllers are gradually replaced by more advanced control strategies such as fuzzy logic and neural networks. In fact, intelligent controllers such as fuzzy logic have a dominant advantage in that they are designed relying almost exclusively on expert experiences and less on the exact mathematical model of the control object. This feature is highly significant in developing control schemes of nonlinear and uncertain systems, which are popular in reality. Theoretically, the structures of fuzzy controllers are highly diverse. Some fuzzy controllers are built based on the principles of traditional controllers like PI, PD, or PID. Figures 3 and 4 depict the structures of PI and PD fuzzy-based controllers, respectively.



Fig. 3 Fuzzy PI controller



Fig. 4 Fuzzy PD controller

Table 1. Fuzzy logic law for PI/PD – like fuzzy logic controllers

	NB	NM	NS	ZE	PS	PM	PB
NB	NB	NB	NB	NB	NM	NS	ZE
NM	NB	NM	NM	NM	NS	ZE	PS
NS	NB	NM	NS	NS	ZE	PS	PS
ZE	NM	NM	NS	ZE	PS	PM	PM
PS	NS	NS	ZE	PS	PM	PM	PB
PM	NS	ZE	PS	PM	PM	PM	PB
PB	ZE	PS	PM	PB	PB	PB	PB



Fig. 5 A 3D illustration corresponding to the fuzzy rule base in Table 1

Theoretically, these controllers' input/output relationship adheres to the classic PI or PD control laws. This means that corresponding to traditional PI or PD controllers, the P and I or P and D coefficients will be adjusted by fuzzy logic without the need for the mathematical model of the control plant. Additionally, these tuning parameters will continuously change depending on the error e(t) rather than being fixed as in the corresponding classic controllers (see Figures 3 and 4).

Fuzzy control laws applied to the PI or PD controllers are also quite diverse, depending on each problem. Fuzzy rules can be constructed in 3x3, 5x5, 7x7 or more. This study will compare the last type of fuzzy rule (7x7) mentioned in Table 1 with the proposed fuzzy logic algorithm. The 3D-illustration corresponding to this rule base is depicted in Figure 5. It can be seen that the fuzzy rule and this 3D graph are entirely symmetrical, similar to the position of the inverted pendulum, which is symmetrical about the vertical axis. 3.3. Propose Lyapunov Theory-Based Fuzzy Logic Controllers

From the linear mathematical model of the inverted pendulum subsystem Equations (7) and (8).

Let 
$$x_1 = \theta$$
 and  $x_2 = \theta$  we get the equations of state:

$$\begin{cases} \dot{x}_1 = x_2 \\ \dot{x}_2 = \frac{u \cos x_1 - (M+m)g(\sin x_1) + ml(\cos x_1 \sin x_1)(x_2)^2}{ml(\cos x_1)^2 - (M+m)l} \end{cases}$$
(10)

Select the semi-positively definite Lyapunov function as:

$$V = \frac{1}{2} \left( x_1^2 + x_2^2 \right)$$
(11)

Take the derivative of the Lyapunov function given in (11), one can be obtained below:

$$\dot{V} = x_1 \dot{x}_1 + x_2 \dot{x}_2$$

$$= x_1 x_2 + x_2 \frac{u \cos x_1 - (M+m)g(\sin x_1)}{ml(\cos x_1)^2 - (M+m)l}$$

$$+ \frac{ml(\cos x_1 \sin x_1)(x_2)^2}{ml(\cos x_1)^2 - (M+m)l}$$

$$= x_2 \left[ \frac{x_1 ml(\cos x_1)^2 - x_1 (M+m)l}{ml(\cos x_1)^2 - (M+m)l} \right]$$

$$+ x_2 \left[ \frac{u \cos x_1 - (M+m)g(\sin x_1)}{ml(\cos x_1)^2 - (M+m)l} \right]$$

$$+ x_2 \left[ \frac{ml(\cos x_1 \sin x_1)(x_2)^2}{ml(\cos x_1)^2 - (M+m)l} \right]$$
(12)

According to Lyapunov's theory, for the system to be stable, it is mandatory to choose u(t) such that  $\dot{v} \le 0$ . Since the denominator is negative, it is recommended that to obtain  $\dot{v} \le 0$  the control signal u(t) must be selected so that the numerator is greater than or equal to zero. It is easy to see that u(t) is chosen to satisfy the stability condition in three cases below:

• If 
$$x_2 > 0$$
 then:  

$$u > \frac{(M+m)[x_1l + g(\sin x_1)] - x_1ml(\cos x_1)^2}{\cos x_1}$$

$$-\frac{ml(\cos x_1 \sin x_1)(x_2)^2}{\cos x_1}$$
(13)

If 
$$x_{2} < 0$$
 then:  

$$u < \frac{(M+m)[x_{1}l + g(\sin x_{1})] - x_{1}ml(\cos x_{1})^{2}}{\cos x_{1}} - \frac{ml(\cos x_{1}\sin x_{1})(x_{2})^{2}}{\cos x_{1}}$$
(14)

• If 
$$x_2 = 0$$
 then:  

$$u = \frac{(M+m)[x_1l + g(\sin x_1)] - x_1ml(\cos x_1)^2}{\cos x_1}$$
(15)

Obviously, it is necessary to select the values of u(t) satisfying conditions (13), (14) and (15) at characteristic points  $(x_1, x_2)$ , as shown in Table 2.

Table 2. Select control signal values u based on equations (13) – (15)

Characteristic Points		Stability	Select the Control	
<b>X</b> 1	<b>X</b> 2	Characteristics	Signal u	
-0.4	-0.8	<i>u</i> < -6.0653	-8.0	
-0.4	0.0	u = -5.9935	-6.0	
-0.4	0.8	<i>u</i> > -5.9217	-4.2	
0.0	-0.8	<i>u</i> < 0	-1.8	
0.0	0.0	u = 0	0.0	
0.0	0.8	u > 0	1.8	
0.4	-0.8	<i>u</i> < 6.0653	4.2	
0.4	0.0	u = 5.9935	6.0	
0.4	0.8	<i>u</i> > 5.9217	8.0	

From the selected values in (Table 2), a Sugeno fuzzy logic inference will be chosen in this study. The two inputs are  $x_1(t)$  and  $x_2(t)$ . The output of the controller is u(t). The fuzzy structure is successfully proposed by choosing the SUM – PROD inference method and the weighted-mean fuzzy logic method. The membership functions and fuzzy logic rule base can be illustrated in Figure 6 and Table 3.

Table 3. A simplified fuzzy logic rule base applied for the proposed hybrid controller

u		X2			
		NE	ZE	РО	
X1	NE	-8.00	-6.00	-4.20	
	ZE	-1.80	0.00	1.80	
	РО	4.20	6.00	8.00	



Fig. 6 Membership functions of the proposed fuzzy logic structure

Remember that these illustrations are highly simplified. There are only three levels for each membership function set. Meanwhile, this work applies nine fuzzy logic rules to balance the pendulum system.

# 4. Simulation Results and Discussions

#### 4.1. Simulation Diagram

The control diagram is built up as shown in Figure 7. Remember that the core control idea is based on fuzzy logic and the Lyapunov stability law, as mentioned in Section 3. In Figure 7, the pole-cart inverted pendulum system model is built following the mathematical equation system expressed in Section 2. Furthermore, a modification has been added to the control system. Three scaling factors, namely  $K_1$ ,  $K_2$  and  $K_3$ . corresponding to three counterparts of a PID controller, have been embedded in the system to enhance the control quality. Significantly, they are determined by using the PSO algorithm, improving the convergence of the control solution. The objective here is formed by satisfying the sum of the absolute values of the position as well, and the rotational angle speed should be minimized. Simulation results are shown below, together with the comments to demonstrate the feasibility of the proposed controller in comparison with other counterparts.

### 4.2. Simulation Results and Comments

To verify the effectiveness of the proposed control scheme, the simulation processes are implemented in MATLAB/Simulink software for the following four types of controllers:

- The proposed hybrid fuzzy logic controller,
- The PD-like fuzzy logic model,
- The PI like fuzzy logic controller and
- The conventional PID regulator.

The above four controllers are first determined using the PSO algorithm (see Figures 8-11). Table 4 indicates optimal values for these scaling factors. Then, they are applied to the IPBCS problem.



Fig. 7 The proposed control diagram for the IPBCS







Fig. 9 Applying the PSO mechanism to find the parameter set of the traditional PID regulator- the convergence



Fig. 10 Using the PSO algorithm to find the parameter set of the PDbased fuzzy logic controller – the convergence



Fig. 11 Running the PSO to find the scaling factors of the fuzzy PI model- the convergence



Table 4. Parameters determined by the PSO algorithm

		Scaling Factors			
		$\mathbf{K}_1$	$\mathbf{K}_2$	<b>K</b> 3	
	Hybrid Fuzzy	0.6556	1.0516	9.4443	
Controllers	PD- Based Fuzzy	0.9708	0.2563	9.4033	
	PI- Based Fuzzy	0.9342	0.2216	4.2646	
_	PID	0.0005	0.001	0.0009	

Now, let us perform the simulation processes using the scaling factors given in Table 4. Four simulation scenarios regarding x(t) and  $\theta(t)$  are also considered. It is noted that  $x_0$  and  $\theta_0$  are two initial values of x(t) and  $\theta(t)$ , respectively.

(i) The first case:  $x_0 = \frac{\pi}{6}; \theta_0 = 0.5 (rad)$ 

Simulation results are plotted in Figure 12 and Figure 13. It is noted that subfigures (b) represent enlarged parts of the total figures (a).

(ii) The second case:  $x_0 = -\frac{\pi}{6}; \theta_0 = 0.8(rad)$ 

Simulation results are plotted in Figures 14 and 15.

(iii) The third simulation case with the following initial parameters:

$$x_0 = -\frac{\pi}{8}; \theta_0 = -0.4 (rad)$$

Simulation results in this case are plotted in the Figure 16 and Figure 17.

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Fig. 12 Simulation results of x(t) in the first case



Fig. 13 Simulation results of  $\theta(t)$  in the first case



Fig. 14 Simulation results of x(t) in the second case







Fig. 16 Simulation results of x(t) in the third case



Fig. 17 Simulation results of  $\theta(t)$  in the third case

# 5. Conclusion and Future Work

This study has proposed a novel control strategy for dealing with the balance of an inverted pendulum system. The hybrid control scheme efficiently integrates the fuzzy logic technique, Lyapunov theory, and the PSO algorithm. Promising numerical simulation results compared with several existing controllers, including PID and PI/PD fuzzy logic counterparts, have demonstrated the dominant effectiveness and applicability of the proposed control approach. Besides, the fuzzy logic rules have been minimized, dramatically reducing the control system's executive time.

Future work will focus on designing a set of more adaptive fuzzy logic rules for various numbers of inverted pendulum systems. In this aspect, practical applications should also be considered to testify to the new control method.

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# Appendix

Table 5. Simulation parameters				
Symbol	Meaning	Value, (unit)		
М	Mass of the cart	1.2 (kg)		
m	Mass of the rod	0.105 (kg)		
1	Length of the rod (mass omitted)	0.11 (m)		
x(t)	Position of the cart	(m)		
$\dot{\mathbf{x}}(t)$	The velocity of the cart	(m/s)		
ÿ(t)	Acceleration of the cart	(m²/s)		
$\theta(t)$	Pendulum angle (upright position)	(rad), (degree)		
$\dot{\theta}(t)$	Rotational velocity of the pole	(rad/s)		
<del>Ö</del> (t)	Rotational acceleration of the pole	(rad <sup>2</sup> /s)		
u(t)	Variable force exerted on the cart	(N)		
K <sub>1</sub> , K <sub>2</sub> , K <sub>3</sub>	Three scaling factors of the fuzzy logic-based position controller	N / A		