

# Design and Optimization of Centrifugal Pump Guide Vanes

G. Kalyan<sup>\*1</sup> K.L.N. Murty<sup>\*2</sup> M-Tech (PhD)

PG Student from Godavari institute of Engineering and Technology Rajahmundry A.P India<sup>\*1</sup>  
Associate Professor in Godavari institute of Engineering and Technology Rajahmundry A.P India<sup>\*2</sup>

## Abstract

In this we proposes a Centrifugal pump design is a most common pump used in industries agriculture and domestic applications. Its impeller design demands a detailed understanding of the internal flow at rated and part load operating conditions. For the cost – effective design of pumps it is very crucial to predict their performance in advance before manufacturing them which requires understanding of the flow behavior in different parts of the pump. In this paper an impeller of a centrifugal pump is designed and modeled in 3D modeling software Pro/Engineer. The present model has 5 blades and angle of blade is  $20^{\circ}$  in this thesis the number blades are chnaged to 6 and 7 and angle od blades is changed to  $30^{\circ}$  and  $50^{\circ}$ . Materials used are steel and aluminum. Structural analysis frequency analysis and CFD analysis are performed. The optimization of the impeller design is done by observing the results obtained from the analysis performed. The results considered are stress frequency velocity pressure flow rates. Analysis is done in Ansys.

## I. INTRODUCTION OF CENTRIFUGAL PUMP

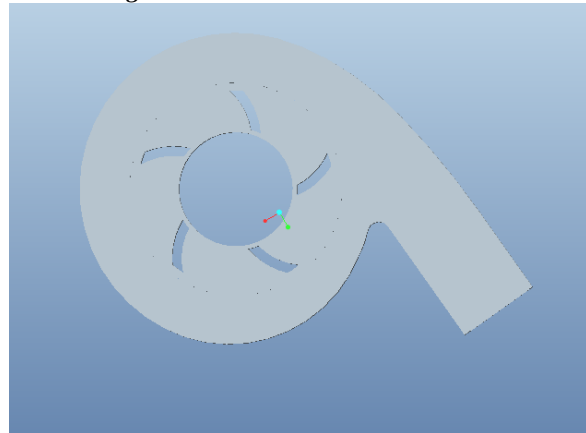
Centrifugal pump is one of the basic and a superb piece of equipment possessing numerous benefits over its contemporaries. The main advantages of a centrifugal pump includes its higher discharging capacity higher operating speeds lifting highly viscous liquids such as oils muddy and sewage water paper pulp sugar molasses chemicals etc. against the reciprocating pumps which can handle relatively small quantity of liquid operating at comparative slower range of speeds that is limited to pure water or less viscous liquids free from impurities limited from the considerations of separation cavitation and frequent choking troubles. The overall maintenance cost of a centrifugal pump is also comparatively lesser due to less wear and tear. While major disadvantage includes vulnerability to a complexities of eddies formations noise and vibrations and inability to generate higher pressures as executed by the reciprocating pumps.

## II. LITERATURE REVIEW

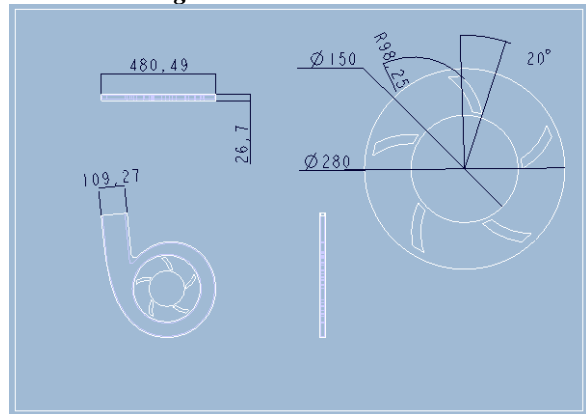
In the paper by E.C. Bacharoudis etal<sup>[1]</sup> describes the simulation of the flow into the impeller of a laboratory pump in a parametric manner. In this study

the performance of impellers with the same outlet diameter having different outlet blade angles is thoroughly evaluated. In the paper by A. Akhras etal<sup>[2]</sup> This paper provides the results of a detailed flow investigation within a centrifugal pump equipped with a vaned diffuser. In the paper by Amit Suhane<sup>[3]</sup> experimental study work carried out on a single stage diffuser type centrifugal pump.

### A. $20^{\circ}$ Angle with 5 Blades

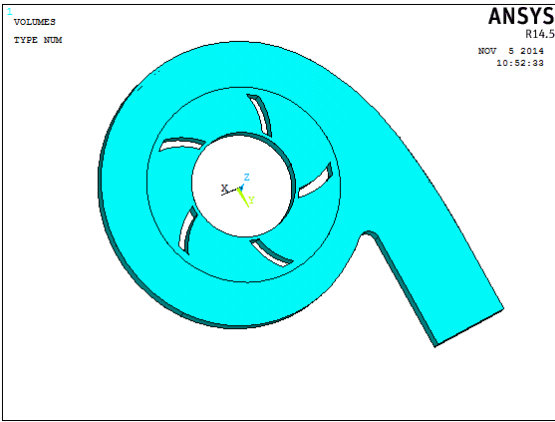


### B. 2D Drawing



**III. ANALYSIS OF CENTRIFUGAL PUMP  
MATERIAL-STEEL STRUCTURAL  
ANALYSIS 5-BLADES WITH 20° ANGLE**

**Imported Model**



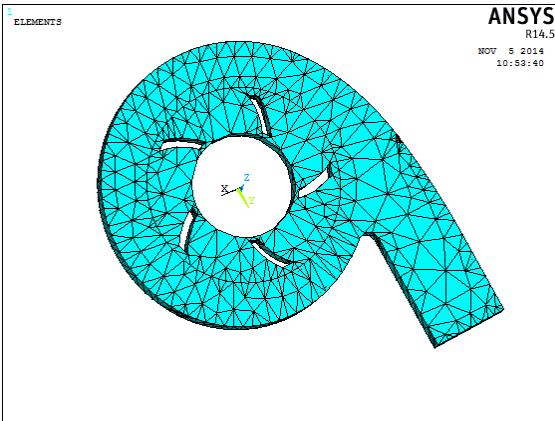
**Steel Material Properties**

Young's modulus=205000MPa

Poisson's ratio=0.29

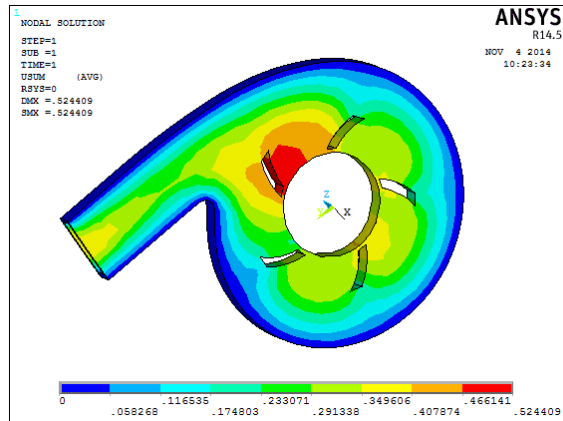
Density=0.00000785kg/mm<sup>3</sup>

**Meshed Model**



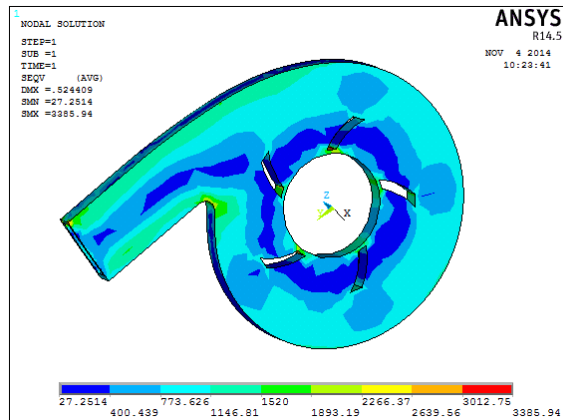
Angular velocity=96.8657rad/sec

**Displacement**



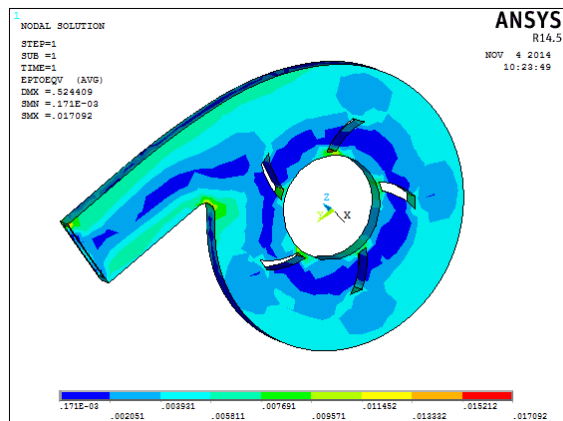
According to this the minimum value for displacement is 0.116535mm and maximum value is 0.524409mm

**Stress**



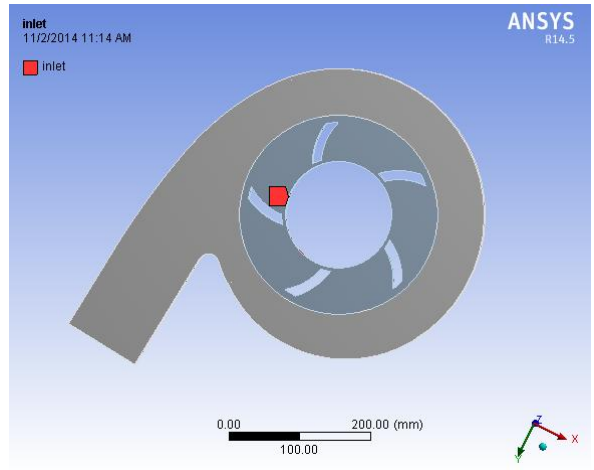
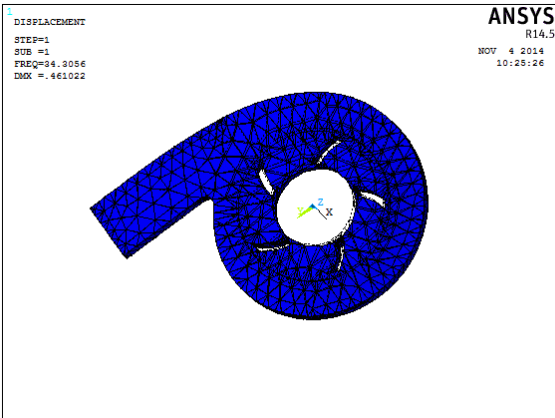
According to this the minimum value for stress is 27.2514 N/mm<sup>2</sup> and maximum value is 3385.94 N/mm<sup>2</sup>

**Strain**



According to this the minimum value for strain is 0.171E-03 and maximum value is 0.17092

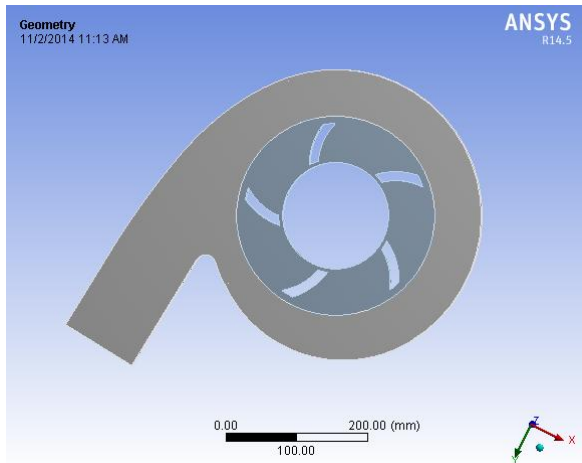
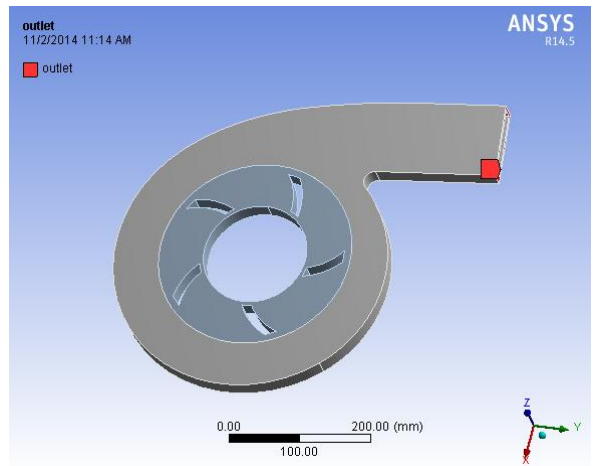
### MODAL ANALYSIS



### CFD ANALYSIS OF IMPELLER IN CENTRIFUGAL PUMP

5 BLADES WITH 20° ANGLE

Mass flow rate – 0.0125m<sup>3</sup>/sec

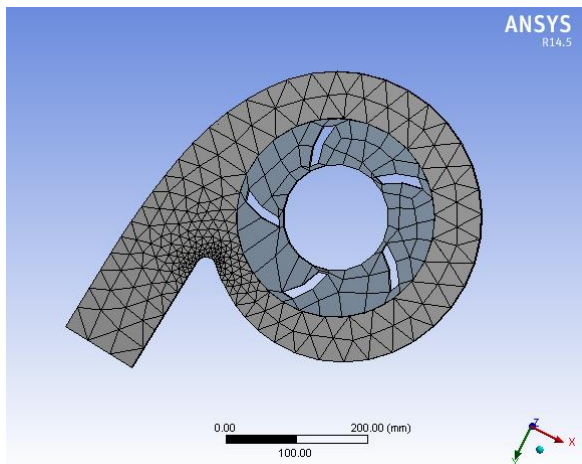


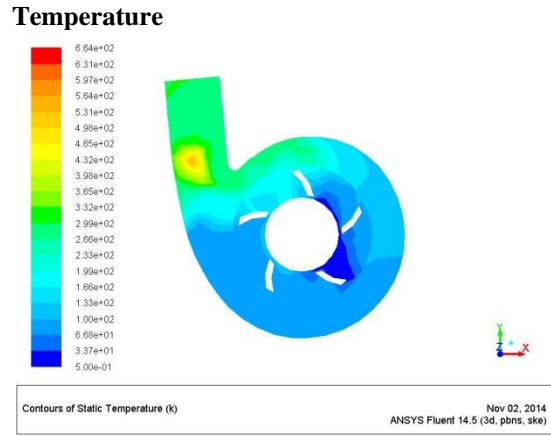
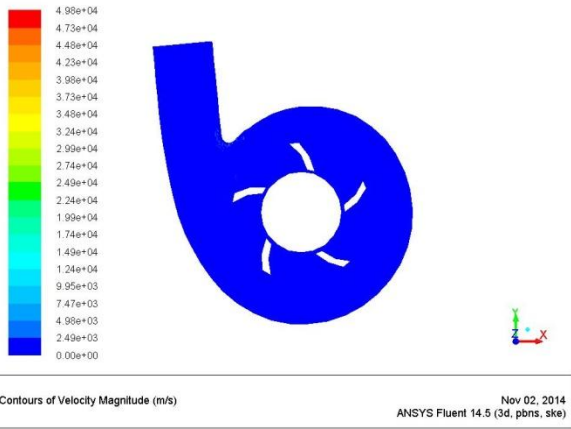
Materials → new → create or edit → specify Fluid material → Air

Boundary conditions → Inlet → Edit

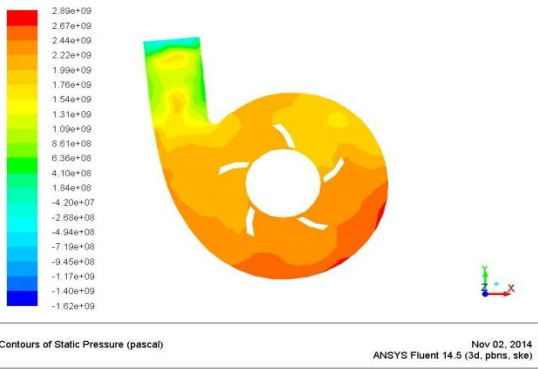
Mass flow rate=0.0125m<sup>3</sup>/sec

**Velocity Magnitude**





Static pressure



IV. RESULTS TABLE STRUCTURAL ANALYSIS

Material	Angle	No of blades	Displacement (mm)	Stress (N/mm <sup>2</sup> )	Strain
Steel	20°	5 blades	0.524409	3385.94	0.17092
		6 blades	0.572838	3593.8	0.017918
		7 blades	0.545191	3583.9	0.017864
	30°	5 blades	0.539388	3591.03	0.017899
		6 blades	0.617447	3591.97	0.017908
		7 blades	0.589036	3590.84	0.017897
	50°	5 blades	0.655216	4305.02	0.021111
		6 blades	0.669112	4033.16	0.019838
		7 blades	0.799257	4520.65	0.02211

Material	Angle	No of blades	Displacement (mm)	Stress (N/mm <sup>2</sup> )	Strain
Aluminum	20°	5 blades	0.544646	1123.36	0.017162
		6 blades	0.59818	1224.57	0.018492
		7 blades	0.570149	1220.3	0.01843
	30°	5 blades	0.557783	1223.37	0.018474
		6 blades	0.640799	1223.24	0.018471
		7 blades	0.616168	1222.41	0.018462
	50°	5 blades	0.673687	1485.76	0.022014
		6 blades	0.689291	1361.32	0.020202
		7 blades	2.40952	4520.65	0.66655

**MODAL ANALYSIS**

**FOR STEEL**

Angle	No of blades	Frequency (Hz)			Deformation (mm)		
		Mode1	Mode2	Mode3	Mode1	Mode2	Mode3
20°	5	34.3856	45.2283	52.2075	0.461022	0.46477	0.463796
	6	36.2225	46.7288	36.0782	0.466547	0.47501	0.477452
	7	36.7201	46.6426	33.6969	0.467985	0.480106	0.488040
30°	5	36.2287	46.7422	55.6214	0.461059	0.47352	0.45923
	6	36.715	46.914	55.8645	0.477664	0.484229	0.476273
	7	26.2807	45.5266	54.2272	0.472289	0.49207	0.509789
50°	5	36.0676	45.6289	54.1309	0.497717	0.522524	0.497724
	6	36.4122	45.6092	52.4406	0.506727	0.505229	0.577268
	7	36.6712	45.0781	53.17	0.524207	0.528044	0.541272

**FOR ALUMINUM**

Angle	No of blades	Frequency (Hz)			Deformation (mm)		
		Mode1	Mode2	Mode3	Mode1	Mode2	Mode3
20°	5	34.4799	45.1982	32.9476	0.795655	0.784801	0.829001
	6	26.5116	46.65	55.7727	0.810369	0.827962	0.824637
	7	26.8096	46.4151	55.2448	0.808226	0.829647	0.826249
30°	5	36.5214	46.7098	33.3613	0.796201	0.827709	0.803937
	6	36.8721	46.5449	55.3549	0.829121	0.54444	0.82275
	7	26.4672	45.628	52.0277	0.821459	0.85566	0.876922
50°	5	26.1095	45.5225	52.702	0.802221	0.90177	0.856429
	6	26.418	45.2292	52.988	0.874635	0.577291	0.996426
	7	21.1204	26.4231	20.6228	0.524207	0.528049	0.541272

**CFD ANALYSIS RESULTS**

Angle	No of blades	Velocity (m/s)	Pressure (Pa)	Temperature (K)
20°	5	4.91e+04	2.87e+09	6.64e+02
	6	3.51e+04	9.73e+09	5.30e+12
	7	1.77e+15	2.00e+11	3.65e+02
30°	5	2.65e+04	2.87e+09	3.08e+02
	6	1.34e+04	1.62e+10	2.94e+02
	7	1.76e+15	4.60e+10	3.32e+02
50°	5	4.04e+04	3.10e+10	2.82e+02
	6	1.27e+04	1.64e+09	4.56e+02
	7	2.61e+04	4.41e+09	3.10e+02

**V. CONCLUSION**

In this paper an impeller of a centrifugal pump is designed and modeled in 3D modeling software Pro/Engineer. The present model has 5 blades and angle of blade is 20° in this thesis the number blades are

chnaged to 6 and 7 and angle od blades is changed to 30° and 50°. Materials used are steel and aluminum. Structural analysis frequency analysis and CFD analysis are performed. Analysis is done in Ansys. By observing the structural analysis results the stresses are increasing

by increasing the number of blades and increasing the angle of blade. When Aluminum material is used the stresses are less than that of steel. By observing modal analysis results the frequencies are reducing by increasing the number of blades thereby the vibrations are reduced. But the frequencies are slightly increasing by increasing the angle of blade. But for aluminum material by increasing the number of blades the frequencies are increasing. By observing CFD analysis results the velocity is decreasing by increasing the number of blades and angle the pressure is increasing and the temperature is decreasing. But for  $50^0$  angle the results are reverses so using angle  $50^0$  is not proposed. So it can be concluded that using Aluminum for impeller is better since the stresses are less. By considering stresses using number of blades 5 and angle  $20^0$  is better but by considering frequencies using number of blades 6 and angle  $20^0$  is better.

#### REFERNECES

- [1] Parametric Study of a Centrifugal Pump Impeller by Varying the Outlet Blade Angle by E.C. Bacharoudis A.E. Filios M.D. Mentzos and D.P. Margaris The Open Mechanical Engineering Journal 2008 2 75-83
- [2] The Internal Flow Investigation of a Centrifugal Pump by Akhras M. El Hajem R. Morel J.Y. Champagne
- [3] Experimental Study on Centrifugal Pump to Determine the Effect of Radial Clearance on Pressure Pulsations Vibrations and Noise by Amit Suhane (IJERA) ISSN: 2248-9622 Vol. 2 Issue4 July-August 2012 pp.1823-1829
- [4] D. Eckardt "Detailed flow investigations within a high-speed centrifugal compressor impeller" ASME Journal of Fluids Engineering vol. 98 pp. 390-402 1976.
- [5] M. W Johnson and J. Moore "The development of wake flow in a centrifugal impeller" ASME Journal of Engineering for Power vol. 102 pp. 382-390 1980.
- [6] A. Kjork and L. Lofdahl "Hot-wire measurement inside a centrifugal impeller" ASME Journal of Fluids Engineering vol. 111 pp. 363-368 1989.
- [7] J.D. Denton "The calculation of three-dimensional viscous flow through multistage turbomachinery" ASME Journal of Turbomachinery vol. 114 pp. 18-26 1992.
- [8] M.W. Dawes "Toward improved through flow capability: the use of three-dimensional viscous flow solvers in a multistage environment" ASME Journal of Turbomachinery 114 pp. 8-17 1992.
- [9] M.V. Casey P. Dalbert and P. Roth "The use of 3D viscous flow calculations in the design and analysis of centrifugal compressors" ASME Journal of Turbomachinery vol. 114 pp.27-37 1992.
- [10] P. Bansod and C.M. Rhie "Computation of flow through a centrifugal impeller with tip leakage" AIAA Paper No 90-2021 1990