

Heat Transfer Investigation of Lagging Materials through Elastic Multi layers

V.Shangari, R.Suji

M.Tech Students, Depart. of Thermal Engineering,
Anja College of Engineering, Sivakasi

Abstract: An original elastic multilayer thermal lagging material is accessible for applications at harsh surroundings as high as 437 K or as low as 125 K. A heat transfer representation is conventional and solved to study heat transfer throughout the material, counting emission; solid heat relocate and gas heat transmit. Most of the flexible defending materials believe the very obligation of these special applications. Such materials ought to be light to decrease the burden, flexible to condense the barriers and energy expenditure, protecting to all kinds of hot, light and emission. Comparison connecting the investigational results and the imaginary calculation shows that the model is practicable to be applied in manufacturing. The temperature allocation of samples with 12, 17, 22, 27, 32 layers, correspondingly, the emission, solid and gas heat transfer of an illustration with 10 layers are analyzed at harsh environment (125 K and 437 K) and the normal circumstance as well. The theoretical thermal analysis provides an energetic training to an optimal intend of such protecting materials. In some way, our study can provide helpful support for further engineering applications of flexible thin-film solar cell.

Key words: flexible, multilayer insulation material, heat transfer, high-low temperature.

I. INTRODUCTION

Fortification of human has been trapped much attention in defensive material investigate, and flexible protecting material is a key factor for standard moving in special occasions to investigate the world and the creation. Most of the flexible defending materials believe the very obligation of these special applications. Such materials ought to be light to decrease the burden, flexible to condense the barriers and energy expenditure, protecting to all kinds of hot, light and emission. It also requires protecting force, organisms or dangerous gas, liquid, and solid contravention. In general, temperature protection together with cold and hot surroundings is the most basic satisfied in protection of human separation. This new coating will cool until it is burning 230 watts from each surface, at which point the whole lot is in stability. The new layer receives 460 watts from the innovative plate. 230 watts is radiated back to the unique plate, and 230 watts to space. The original outside still radiates 460 watts, but gets 230 back from the new layers, for a net defeat of 230 watts. So in general, the radiation losses have been condensed by half by totaling the supplementary layer.

The bendable material has soft, light and good lagging performance and it is adequate to form the thermal defense materials independently. Consequently, the most thing people desire to accomplish and study constantly is that the use of

structural intend and materials arrangement to improve the thermal isolation routine, to reduce the weight and breadth, and to increase the suppleness. So the exploration in this paper is listening carefully on the flexible multilayer thermal lagging material for high/low high temperature confrontation.

i. The constitution of Flexible Multilayer Insulation Material

The evacuated material consists of external fabric, inner fabric, and central point multilayer lagging systems. The middle multilayer system consists of a large number of exceedingly thoughtful shields, frequently between 6 and 12 micrometers of padding breadth, estranged from each other by thin non-conducting spacers. The shields are normally made out of polyester (PET) films (typically 6 to 12 micrometers). For shimmering, the shields are covered with vacuum- deposited aluminum with a thickness between 0.03 and 0.05 micrometers. The ordinary materials for the spacer are polyester netting, and bonding PET films are added to defend the aluminum film and to decrease heat transmit.

ii. Mathematical Models for Thermal investigation

The heat transfer machinery of the multilayer lagging material can be enormously compound, and can be modeled with the fractal imitative, and the theoretical examination of thermal insulation for low/high warmth insulation is of

financial concentration, and a biomimic intend of multi-scale materials. At this point, the heat transfer during the middle multilayer organization is discussed. The heat transfer through the middle system consists of solid heat transfer, gas relocate and emission. Sketch of the warmth allocation for multilayer thoughtful shields.

iii. The Heat Flux during the thoughtful Shields

The thoughtful shield can be viewed as multi-level wall. So in a steady-state circumstance the heat flux (Q) through i insightful shields can be printed with respect to the confidence of the thermal conductivity of the hotness:

$$Q = \frac{t_i - T_i}{R_f} = \frac{A(t_i - T_i)}{\frac{\delta_1 + \delta_3}{c_{Pet} + d_{Pet} \left(\frac{T_i + t_i}{2}\right)^{e_{Pet}}} + \frac{\delta_2}{c_{Al} + d_{Al} \left(\frac{T_i + t_i}{2}\right)^{e_{Al}}}} \quad (1)$$

Where R_f is the thermal confrontation, δ_1 – the breadth of PET, δ_2 – the thickness of Al, δ_3 – the thickness of PET (bonding layer), and A – the area of the project. c_{Pet} , d_{Pet} , e_{Pet} , c_{Al} , d_{Al} , e_{Al} are the constants for experimental function of thermal conductivity, T_i and t_i are the margin temperatures of i thoughtful defend.

iv. The Heat Flux among the Reflective Shields in Multilayer Thermal lagging Material

Heat transfer connecting reflective shields includes solid heat transfer, gas heat transfer and thermal emission that are in equivalent. The solid heat transfer includes heat transmission through spacers and contact heat relocates between spacer and protect that are series. As well as, the gas heat transfer includes gas convection and transmission. Nonetheless the amount of contact heat transfer connecting spacer and shield is changeable and gas convection is so small that they are deserted.

v. The Solid Heat Conduction throughout Spacer

Consider the confidence of the thermal conductivity of the temperature for the spacers, the heat instability through space among $i-1$ shield and i shield can be obtained.

$$Q_s = \left[c_s + d_s \left(\frac{t_{i-1} + T_i}{2}\right)^{e_s} \right] \frac{A_s (T_i - t_{i-1})}{\delta} \quad (2)$$

Where Q_s is the heat flux throughout space among $i - 1$ shield and i shield, c_s , d_s , e_s are the constants of the spacer, A_s is the area of the spacer, and δ is the breadth of the spacer.

vi. The emission Heat Transfer among the Reflective Shields

In high vacuum surroundings, the heat transfer through thoughtful shields not moving each other is simply emission heat transfer Q_r , consequently, the radiation heat flux is the smallest amount of the heat flux for this multilayer scheme, which cannot be reached in practical submission. But the smallest heat flux can be worn as a scale for assessment to a basically realized multilayer lagging arrangement.

For the emission heat flux involving two adjacent reflective shields the subsequent appearance is valid, considering the production of the reflective shields surfaces in a good estimate confidence of the hotness.

$$Q_g = \frac{A_g (T_i - t_{i-1})}{\delta} \left\{ \left[\alpha \left(\frac{\gamma+1}{\gamma-1}\right) \left(\frac{R}{8\pi M_g \frac{T_i - t_{i-1}}{2}}\right)^{0.5} P\delta \right]^{-1} + \left[\lambda_0 + h \frac{t_{i-1} + T_i}{2} \right]^{-1} \right\} \quad (3)$$

$$Q_r = \frac{A_g \sigma (T_i^4 - t_{i-1}^4)}{\frac{1}{a_1 t_{i-1}^{b_1}} + \frac{1}{a_2 T_i^{b_2}} - 1} \quad (4)$$

Where a_1 , b_1 , a_2 , and b_2 are constants for the reflective shields, σ is the Stefan-Boltzmann invariable ($5.67 \cdot 10^{-8} \text{ W/m}^2 \text{ K}^4$).

For the total heat flux among two adjacent shields can be articulated:

$$Q = \frac{A_g (T_i - t_{i-1})}{\delta} \left\{ \left[\alpha \left(\frac{\gamma+1}{\gamma-1}\right) \left(\frac{R}{8\pi M_g \frac{T_i - t_{i-1}}{2}}\right)^{0.5} P\delta \right]^{-1} + \left(\lambda_0 + h \frac{t_{i-1} + T_i}{2} \right)^{-1} \right\} + \frac{A_g \sigma (T_i^4 - t_{i-1}^4)}{\frac{1}{a_1 t_{i-1}^{b_1}} + \frac{1}{a_2 T_i^{b_2}} - 1} + \left[c_s + d_s \left(\frac{t_{i-1} + T_i}{2}\right)^{e_s} \right] \frac{A_s (T_i - t_{i-1})}{\delta} \quad (5)$$

The heat flux during the material, the boundary temperatures, the breadth of the shields and spacers and supplementary parameters can be obtained by the experiments. Then the temperature allocation of the whole material and the concrete heat

flux, gas heat flux and emission heat flux throughout every layer can be predictable. The temperature sharing of shields can be intended by Newton iteration and dichotomy technique. The solid heat transfer fluctuation, the gas transfer flux and radiation relocate flux connecting adjacent reflective shields can be designed by correspondence.

II. RESULTS AND DISCUSSION

As we mentioned, the computation does not claim to be talented to predict accurately heat transfer principles. So the numerical consequences should only give ideas for humanizing the lagging excellence. For specimens of 12, 17, 22, 27, 32 layers thermal lagging materials. Generally, in the sense the deviations between the premeditated outcome and the experimental ones are not considerable. And the calculated principles are a little lower than investigational values. for the reason that there are other heat relocate modes which are deserted in the numerical model throughout the multilayer materials, this indicates that the numerical representation which is accessible is this paper turns to be a good demonstration of the heat flux through the multilayer substance.

Figure 1 shows the high temperature distributions of thoughtful shield with different layers, for specimens with 12, 17, 22, 27, 32 layers of the middle multilayer arrangement. It can be shown that the quantity of layers (N) has great authority on the temperature allocation in multilayer thermal lagging substance. The smaller is the integer of the layers, the more intensely the inner temperature varies. The temperature incline increases with the diminish of the warmth.

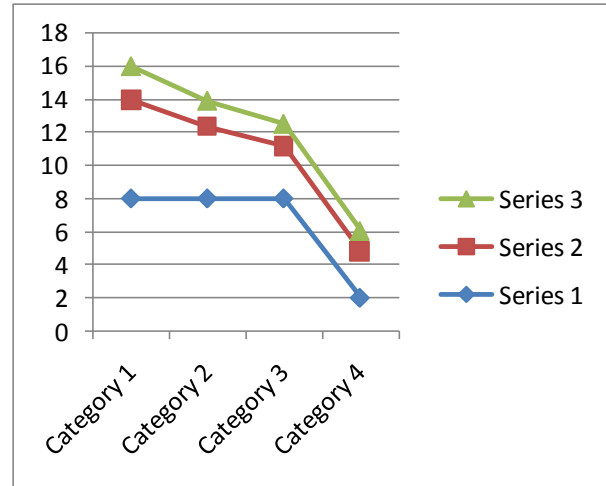
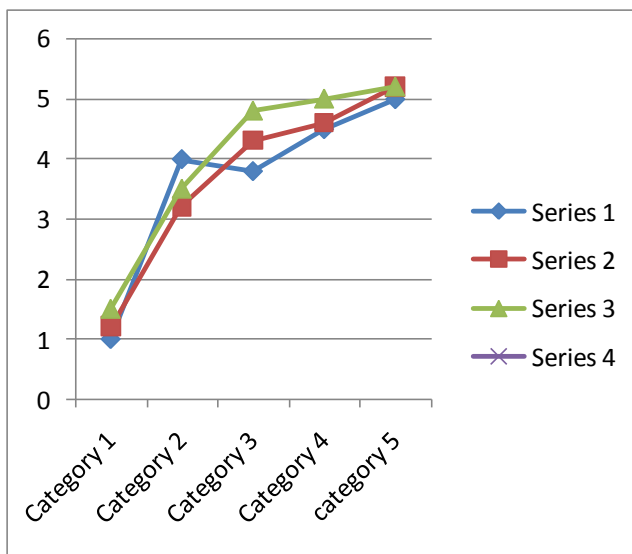


Figure 3. Temperature distributions of reflective shields with different layers from cryogenic temperature (123 K) to normal temperature (273 K) and high temperature (433 K) to normal temperature.

The gas heat transfer fluctuation is almost the equivalent from layer to layer. So, the solid heat relocates payment decreases while the emission contribution increases. The solid heat transport is the main part impending the cold temperature (125 K). The emission heat transfer is the main part forthcoming the ordinary temperature (275 K). The gas heat transfer involvement is tiny from layer to layer.

From the standard temperature (275 K) to hot temperature (435 K), the solid heat reassigns flux decreases with the augment of the temperature. The gas heat relocate flux is small and is approximately the same beginning layer to layer. There is a little augment for the ratio of the energy heat flux and the emission heat transfer is the main part connecting hot temperature and ordinary high temperature.

III. CONCLUSIONS

The errors connecting investigational heat fluxes and the intended results by representation in this paper are below 15%. Comparing investigational results and theoretical calculations, it is shown that the hypothetical calculations are dependable with the considered heat flux trends. According to the model for the computation of temperature allocation and emission, solid, gas transfer flux from high/low temperature to standard high temperature, solid transfer flux decreases while energy heat transfer flux increases with the augment of the temperature from cold to ordinary temperature, and radiation relocate is the main part in heat transfer from high to normal

temperature. In conclusion, the model is significantly useful to design and progress the properties of the multilayer substance.

Nevertheless, there are differences connecting theoretical calculations and investigational results which can be completed that there is still room for enhancement. There are three reasons for the differences as follows. Foremost, the non-uniform arrangement of material itself and the defects led to the subordinate theoretical value and it is not consistent. Succeeding, in different temperature environment, there are changes in the thermal conductive behavior of the substance, which is the difficulty of material itself and dimensions. Third, all the distribution and expression of heat transfer are measured as the thermal resistances in the theoretical inference.

REFERENCES

1. Jin-Jing Chena, B, Zheng Guoa, B, and Wei-Dong YU, Heat Transfer Analysis of Insulation Materials with Flexible Multi layers, *Thermal Science*, Year 2013, Vol. 17, No. 5, Pp. 1415-1420.
2. Chen, J. J., Yu, W. D., A Numerical Analysis of Heat Transfer in an Evacuated Flexible Multilayer Insulation Material, *Journal of Thermal Analysis and Calorimetry*, 44 (2010), 18, pp. 2191-2203.
3. Chen, J. J., Yu, W. D., Concept for High-low Temperature Resistance Arrangement within Multilayer Flexible Composite, *Journal of Composite Materials*, 101 (2010), 3, pp. 1183-1188.
4. Spinnler, M., *et al.*, Studies on High-temperature Multilayer Thermal Insulations, *International Journal of Heat and Mass Transfer*, 47 (2004), 6-7, pp. 1305-1312.
5. Hofmann, A., the Thermal Conductivity of Cryogenic Insulation Materials and Its Temperature Dependence, *Cryogenics*, 46 (2006), 11, pp. 815-824.
6. Krishna prakas, C. K., Radiation Heat Transfer in a Participating Medium Bounded by Specular Reflectors, *Heat Mass Transfer*, 25 (1998), 8, pp. 1181-1188.